

		W	/INTER- 17 EXAMINATION	N		
9	Subject	Name: Theory of Structures	Model Answer	Subject Code:	17422	
Impo	ortant I	nstructions to examiners:				
1) The sche	answers should be examined by eme.	key words and not as wo	ord-to-word as given	in the model answ	wer
2	unde	model answer and the answer wri erstanding level of the candidate.		•		
3		language errors such as gramm icable for subject English and Con		ould not be given m	ore Importance (Not
4		e assessing figures, examiner ma				
		es drawn by candidate and model e drawn.	I answer may vary. The ex	caminer may give cree	dit for any equival	ent
5		dits may be given step wise for n	numerical problems. In so	me cases, the assur	ned constant valu	ues
		vary and there may be some diffe				
6		ase of some questions credit ma	ly be given by judgement	t on part of examine	r of relevant answ	wer
7		ed on candidate's understanding.	aradit may be given to	ony other program k	and an aquival	ont
7) FOI cond	programming language papers,	credit may be given to	any other program t	ased on equival	
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-	Sub		Answer			Markir
-	Q.		Answer			Markir g
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-	Q.		Answer			Markir g
lo.	Q.	Attempt any SIX of the following				Markir g Schem
lo. 2.1	Q. N.					Markir g Schem e
lo. 2.1	Q. N. (A)	Attempt any SIX of the following	j.	ch the line of action o		Markin g Schem e
No. Q.1	Q. N. (A) A)a)	Attempt any SIX of the following Define core of the section. It is the portion of a section arou so as to produce only compressiv	, und the center within whice ve stress is called as core o			Markir g Schem e (12)
-	Q. N. (A) A)a)	Attempt any SIX of the following Define core of the section. It is the portion of a section arou so as to produce only compressiv	und the center within whic	of the section.		Markin g Schem e (12) 01
lo. 2.1	Q. N. (A) A)a)	Attempt any SIX of the following Define core of the section. It is the portion of a section arou so as to produce only compressiv	, und the center within whice ve stress is called as core o			Marki g Schen e (12) 01

Q.1	A)b)	Define slope and deflection of a beam.				
	Ans	Definition of Slope of beam: The slope at any point on the elastic curve of the beam is defined	01			
		as the angle in radians that the tangent at that point makes with the original axis of the beam.	Mark			
		It is measured in radians				
		Definition of deflection of beam: when a beam is loaded, the beam is deflected from its	01			
	original position in the direction perpendicular to its longitudinal axis. Then displacement of					
	beam measured from its neutral axis from unloaded condition of the beam to loaded condition					
		is called deflection of beam.				
		OR				
		The deflection at any point on the axis of the beam is the distance between its positions before				
		and after loading.				
Q.1	A)c)	Write the value of max. slope and deflection in case of simply supported beam loaded with				
		udl over entire span.				

1

b/3

y¦ b Rectangular Column d/4

Circular Column

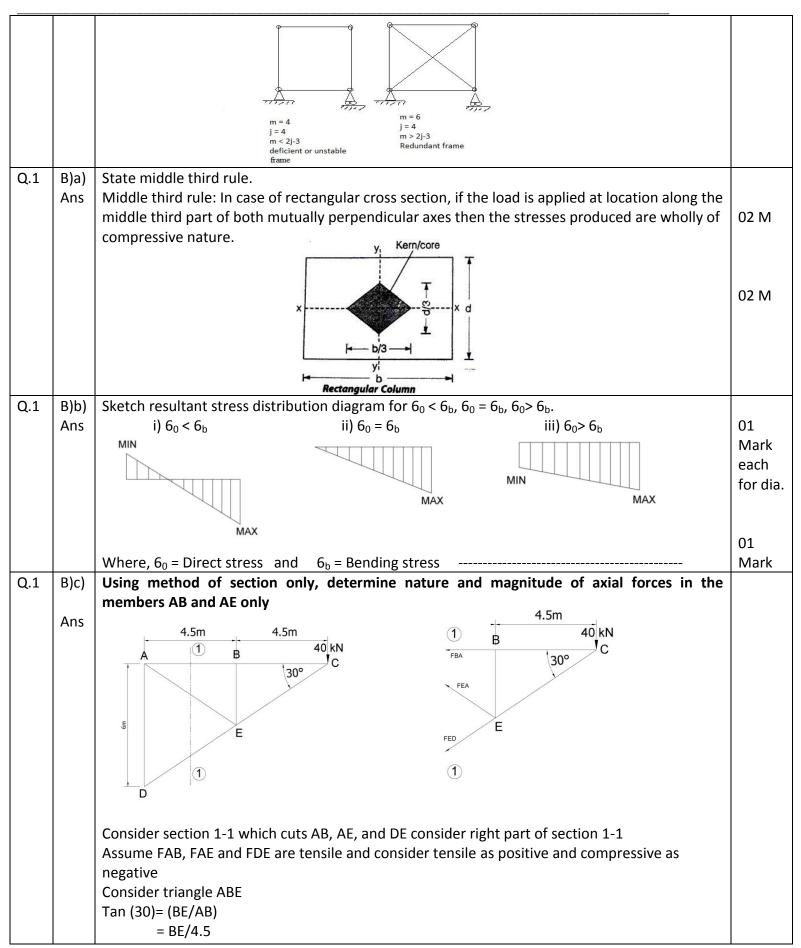
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Mark



	Anc	Signs at the ends of S S here $= \frac{1}{2} - \frac{3}{24}$	01 14
	Ans	Slope at the ends of S.S. beam = $(\Theta)=wL^3/24EI$	01 M
		Deflection at the centre= $y_{max}=y_{centre}= 5/384 \text{ wL}^4/\text{EI}$	01 M
		Where	
		w= rate of loading.(KN/m)	
		L= leangth of beam(m)	
		E= modulus of elasticity(N/mm ²)	
		I= moment of inertia of a beam mm ⁴	
Q.1	A)d)	State the boundary conditions for simply supported beam using deflected shape.	
	Ans	Boundary conditions of simply supported beam (slope exists but deflection is zero)	
		1) slope (Θ)= dy/dx \neq 0	
		2) deflection = $y=0$	
		W	
		K X Beam	
		θ_A $CD = y_A$ θ_B	01
		Tangent	01
		R_A = Reaction force at support A = W/2	Mark
		R_B = Reaction force at support B = W/2	01
		θ_{A} = Slope at support A	Mark
		$\theta_{\rm B}$ = Slope at support B	IVIAIK
Q.1	A)e)	Define fixing and fixed beam	
Q.1	Ans	Fixing: - When the ends of the beam are firmly built in the support so as the slopes at the	01 M
	AIIS	support become zero i.e tangent to the deflected curve at support will be zero.	OT IVI
		Fixed beam: - A beam whose end supports are such that the end slopes remain zero is called a	01 M
		fixed beam.	UTIVI
Q.1	A)f)	Define distribution factor and carry over factor.	
Q.1	Ans	Distribution factor:- it is the ratio of relative stiffness of a member to the total stiffness of all	01 M
	AIIS	the members meeting at a point.	OT IVI
		Carry over factor:- it is the ratio of moment produce at a joint to the moment applied at the	01 M
		other joint without displacing it.	
Q.1	A)g)	Write the concept of carry over factor	
~	Ans	Carry over factor:- it is the ratio of moment produce at a joint to the moment applied at the	01 M
		other joint without displacing it.	
		1) The beam fixed at one end and simply supported at other end, the carry over factor is ½.	01 M
		2) The beam simply supported at both ends, the carry over factor is zero.	
Q.1	A)h)	Define with sketch deficient frame and redundant frame	
	Ans	Deficient frame	
		Assume, n = number of members, j= number of joints. If the number of members are less than	01 M
		the required number of members ($n < 2j-3$) then the corresponding frame is called as deficient	
		frame.	
		Redundant frame	
		Assume, n = number of members, j= number of joints. If the number of members are less than	01 M
		the required number of members ($n > 2j-3$) then the corresponding frame is called as deficient	
		frame.	
	L		







		E= 2.596 ≈2.6m	
		Consider the right part of section 1-1 in equilibrium taking moment at joint E We get	
		$\Sigma M_{\rm E} = -F_{\rm BA} \times 2.6 + 40 \times 4.5$	
		$F_{AB} = 69.23KN \text{ (tensile)}$	02 M
		To find F_{AE} and F_{DE} using condition of equilibrium	02 101
		$\Sigma fx = 0$	
		$-F_{BA} - F_{EA}\cos 30 - F_{ED}\cos 30 = 0$	
		$F_{EA} = F_{EA} \cos 30 = F_{ED} \cos 30 = 0$ $F_{EA} \cos 30 = -69.23$ A	
		$\Sigma_{\rm EAC0350} = 1_{\rm EDC0350} = -03.23$	
		$-40 + F_{EA}\sin 30 - F_{EDsin}\cos 30 = 0$	
		$F_{EA}sin30 - F_{ED}cos30 = 40$ B	
		Solving equations A and B	
		We get	
		F _{EA} = 0.003KN≈ 0KN (tensile)	02 M
		F _{EA} = 0.005KN~ 0KN (tensile) F _{ED} = - 79.969KN≈ -80KN(compressive)	02 101
Q.2	a)	A tie rod of rectangular section having 15mm thickness it carries load of 200KN acts at an	
Q.2	a)	eccentricity of 10mm along a plane bisecting thickness. Calculate the width of section if	
		maximum tensile stress shall not exceed 100MPa.	
	Ans	maximum tensile stress shall not exceed 100MPa.	
	AIIS		
		b = ?	
			01M
			UTIVI
		e=10mm	
		Given:-	
		D=15mm	
		e= 10mm	
		load line bisecting the thickness	
		maximum tensile stress (σ_{max}) = 100 MPa = 100 N/mm ²	
		Since the load is tensile on the right side of YY axis, the maximum tensile stress will occur on	
		the right face of section face BC	
		Let 'b' be the minimum width of the rod	
		If the load is eccentric about YY axis	01 M
		$\sigma_{max} = P/A + M/Zyy = (P/A) + [P.e/(db^2/6)]$	
		$100 = 200 \times 10^3 / b \times 15 + [(200 \times 10^3 \times 10)/(15 \times (b^2/6)]$	
		$100 = 1.3333 \times 10^4 / b + 8 \times 10^5 / b^2$	01M
		$b^2 - 1.3333 \times 10^2 b - 8 \times 10^3 = 0$	01101
		on solving we get	01 M
		b=178.23mm	OTIVI
		N-1/0.42111111	



Q.2	b)	A rectangular column of size 0-35m x0.25 m carries an eccentric load of 150 KN. The load acts	
Q.2	5)	at 0.15m from c.g. of the section on axis bisecting the shorter side. Determine resultant	
		stress at the base and draw stress distribution diagram.	
	Anc	-	
	Ans	Given:- 150 kN 150 mm	
		b= 0.35m = 350mm	1/204
		d= 0.25m= 250mm	1/2M
		P = 150 KN	4 /21.4
		e= 150mm	1/2M
		load line bisecting shorter face i.e. thickness	1/2M
		area (A)= b xd = $350 \times 250 = 87500 \text{ mm}^2$	
		direct stress (σ) = P/A =150 x10 ³ / 87500 =1.71 N/mm ² (comp)	1 M
		bending stress (σ b) = M/Z=P.e/Zyy =150 x10 ³ x 150/ ((250 x350 ²)/6) =	
		4.41 N/mm ² (Comp. at right face and Tensile at left face)	1 M
		$\sigma_{max} = \sigma_0 + \sigma_b = 1.71 + 4.41 = 6.12 \text{ N/mm}^2 \text{ (comp)}$	for
		$\sigma_{min} = \sigma_o - \sigma_b = 1.71 - 0.44 = -2.7 \text{ N/mm}^2 \text{ i.e. } 2.7 \text{ N/mm}^2 \text{ (Tensile)}$	diagra
		6.12 mPa	m
Q.2	c)	A hollow C.I. column of external diameter 300mm and internal diameter 250mm carries an	
~	~,	axial load of 'W' KN and load of 100KN at an eccentricity of 175mm. calculate minimum value	
		of W so as to avoid tensile stresses.	
	Ans	Given	
		External diameter D= 300mm	
		Internal diameter d= 250mm	
		Axial load = W KN	
		Eccentric load (P)= 100 KN	
		Eccentricity e= 175mm	
		Avoid tensile stress i.e. assume no tension condition i.e	
		direct stress (σ o)= bending stress (σ b)	
		To find	
			1M
		Axial load W Area (A) = $\pi/4(D^2-d^2) = \pi/4(300^2-250^2) = 21.6 \times 10^3 \text{mm}^2$	1141
		Direct stress (σo) = (W+P)/A =[W + 100 x10 ³ / 21.6x10 ³ mm ²] (1)	1M
		Bending stress (σ b) = M/Z=P.e/Zyy	
		$=\{100 \times 10^{3} \times 175 / [\pi/32((300^{4} - 250^{4}) / 300)]\}$	1M
		bending stress (σ b) = 12.75 N/mm ² (2)	
		to avoid tensile stress we have to assume no tension condition	1M
		i.e	TIM
		Direct stress (σο)= Bending stress (σb)	
		equating (1) and (2)	
		$[(W + 100) \times 10^3 / 21.6 \times 10^3] = 12.75$	
		We will get W= 175.4 kN	
0.2	d)		
Q.2	d)	A cantilever beam of span 1.8m carries 30 KN/m udl over entire span. if deflection at free end	
	۸	is limited to 25mm, determine the elastic modulus of material I=1.3x10 ⁸ mm ⁴ .	
	Ans	Given	
		L= 1.8m	
		W= 30 KN/m	1



		y = 25 mm I = 1.3x10 ⁸ mm ⁴ 30 kN/m	
		For a cantilever beam carrying UDL over entire	
		span A 25mm	2M
		The deflection is given by the formula	2111
		$y = wL^4/8EI$ 1.8m	1M
		$25 = (30 \times (1.8 \times 10^3)^4)/(8 \times E \times 1.3 \times 10^8)$	1141
		On solving we get	1M
		$E=12.112 \times 10^3 \text{ N/mm}^2$	1111
Q.2	e)	A beam of span 3m is simply supported and carries udl of 'W' N/m if slope at the ends is not	
Q.2	C)	to exceed 1 [°] , find the maximum deflection.	
	Ans	Θ = slope at the end =1° =(1 x π /180) radians = 0.017 rad	1/2M
	Alls	Θ = slope at the end simply supported and carries udl on entire span is given by =wL ³ /24 El	1/21vi 1M
		$0.017 = (w/El)x (L^3/24)$	TIM
		(w/EI) = 0.0151	
		To find maximum deflection for simply supported and carries udl (for downward deflection) $Y_{max} = [5/384(wL^4/EI)]$	02M
		$Y_{max} = 5L^4/384 \text{ (w/EI)}$	02101
		$Y_{max} = -5L^{4}/384 \times 0.0151$	
0.2	L)	Y _{max} = 15.9 mm ≈ 16mm	
Q.2	f)	Clapeyron's theorem of three moments with neat sketch and give meaning of each term	
	Ans	For a two span continuous beam	
		having uniform moment of inertia, (a)	
		supported at ends A, B and C	
		subjected to any external loading , the support moments MA. MB and $dx = M_x$	
		the support moments MA, MB and $dx M_x$ MC at the supports A,B and C	
		respectively are given by the + + (b)	
		$M_{A}L_{1}+ 2MB(L_{1}+L_{2})+MCL_{2} = \overline{x_{1}}$	
		$M_{L_1} + 2M_{L_1} + 2M_{L_1} + M_{L_2} - $	114
		$-(6a_1x_1/L_1+6a_2x_2/L_2)$ Free B.M.D Where	1M
		dx M' MP	1 \ 1
		L_1 = length of span BC	1M
		L_2 = length of span BC 4 + M_c (c)	
		a ₁ = area of free BMD for the span AB (figure b) $\overline{x_1}'$ $\overline{x_2}'$	
			2M for
		a ₂ = area of free BMD for the span Fixed B.M.D BC (figure b)	dia.
			ula.
		x_1 = distance of C.G. of free BMD over the span AB from Left end A M_A + ve - ve + ve + ve M_C (d)	
		over the span AB from Left end A x_2 = distance of C.G. of free BMD	
		over the span BC from right end C	
0.2	2)	A contilouer beam 2 m long corruing und of intensity 6 kN/m over full longth. Coloulate the	
Q.3	a)	A cantilever beam 2 m long carrying udl of intensity 6 kN/m over full length. Calculate the depth of the beam if may, deflection is limited to 5 mm and depth to width ratio is $2.5 - 2 \times 10^5$	
		depth of the beam if max. deflection is limited to 5 mm and depth to width ratio is 2. $E = 2 \times 10^5$	
	4.000	mPa.	
	Ans		



		6 kN/m	
		2 m	
			01 M
		$Y_{max} = (wl^4) / (8El)$	
		$5 = (6 \times 2000^{4}) / (8 \times 2 \times 10^{5} \times I)$ I = 12 x 10 ⁶ mm ⁴	01 M
		$I = bd^{3} / 12$	
		$12 \times 10^6 = b \times (2b)^3 / 12$ (d = 2b)	01 M
		b = 65.136 mm d = 2 x 65.136 = 130.27 mm	01 M
Q.3	b)	A simply supported beam carries udl of 4KN/m over entire span of 4m find deflection at mid	
Q.15	~,	span in terms of El.	
	Ans	W= 4KN/m	
		L= 4m EI= flexural Rigidity (kN-m ²)	
		The formula for the deflection of simply supported beam carrying udl over entire span is given	
		by	
		$Y_{max} = (5 \times w \times L^4) / 384EI$	2M
		$Y_{max} = (5 \times 4 \times 4^4)/384EI$ $Y_{max} = 13.33/EI m.$	2M
Q.3	c)	A fixed beam AB of span 4m carries a point load of 80 KN at its centre. Find fixed end	
		moments by using the first principle and draw	
	Ans	SF and BM diagrams 80 kN Simply supported bending moment at mid-span	
		$= WL/4 = 80 \times 4 / 4 = 80 \text{ kN-m.}$	
		Due to symmetry, $M_{AB} = M_{BA}$	1M
		Area of S. S. B. M. Dia. = $a_1 = 0.5 \times 4 \times 80 = 160$	114
		Area of simply supported bending moment	1M
		diagram = Area of fixed end moment diagram	
		a ₁ = a ₂ S. S. B. M. D.	
		160 = M _{AB} x 4 MAB (2) MBA Hence M _{AB} = 40 kN-m And M _{BA} = 40 kN-m	1M for diagra
		F. E.M. D.	m
		80 kN-M	
		40 kN-M 40 kN-M	
			01 M
		B.M. D.	for
		40 kN	BMD &
			SFD
		S.F. D. 40 kN	

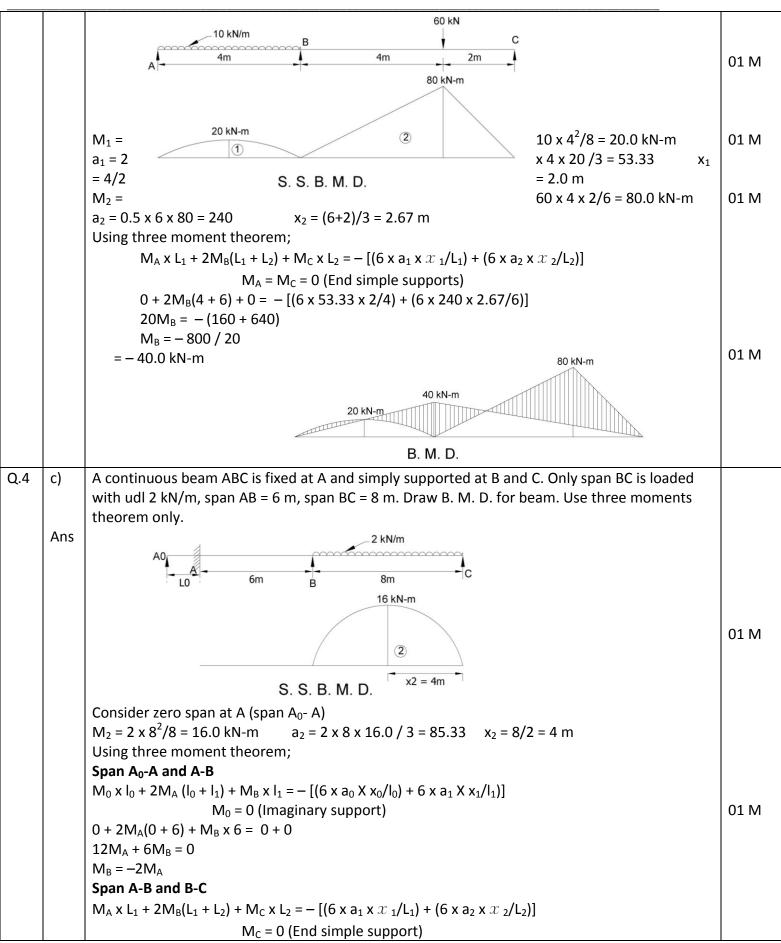


Q.3	d) Ans	 State any two advantages and dis advantages of fixed beam over simply supported beam Advantages of fixed beam over simply supported beam: Due to end fixity ,end slope of a fixed beam is zero. A fixed beam is more stronger,stiffer and stable. For same span and loading,fixed beam has lesser value of Bending moment. Smaller moment permits smaller sections and there is saving in beam material. Fixed beam has lesser deflection for same span and loading as compared to S.S. beam Disadvantages of fixed beam over simply supported beam: A little sinking or settlement of support induces additional moment at each support. secondary stresses are develop due to temperature dynamic loading may disturb the fixity Using method of joints, find nature and magnitude of forces in AE and DE in frame as shown 	1M each for any two 1M each for any two
Q.3	e) Ans	$\begin{array}{c} 15 \text{ kN} & 10 \text{ kN} & 20 \text{ kN} \\ \hline & & & & & & & \\ \hline & & & & & & & \\ \hline & & & &$	02 M
		joint A Assuming forces tensile in nature. Using condition of equilibrium	02 M



		Σ fy = 0 = -15 + 45 - F _{AE} sin45 = 0	
		F _{AE} = 30KN (Tensile)	
		Member Force Nature	
		AE 30 kN Tensile	
		DE 50 kN Compressive	
Q.3	f)	What is meant by analysis of frame? Write the assumptions used for analysis	
		Analysis of frame-	
	Ans	To calculate the magnitude and nature of forces of the members of the frame (perfect frames)	02
		using equilibrium conditions is called analysis of frames.	Marks
		Assumptions made for analysis of frame:-	
		1) the frame is perfect frame	
		2) All members are hinged or pinned connected at the ends.	
		3) the loads are acting only at the joints	02 M
		4)self-weight of the member is neglected	
Q.4		Attempt any FOUR of the following.	(16)
Q.4	a)	A beam ABC is simply supported at A, B and C. Span AB and BC are of length 4 m and 5 m	
		respectively. AB carries a point load of 20 kN at center. BC carries a udl of 10 kN/m over entire	
		span. Calculate support moment at B using theorem of three moments.	
	Ans		
		20 kN 10 kN/m	
	-	4m 5m	
		31.25 kN-m	
		20 kN-m	
			01 M
		x1=2m S. S. B. M. D. x2=2.5m	
		0.0.0.0.	
		$M_1 = 20 \times 4/4 = 20.0 \text{ kN-m}$ $a_1 = 0.5 \times 4 \times 20 = 40$ $x_1 = 4/2 = 2.0 \text{ m}$	01 M
		$M_2 = 10 \times 5^2/8 = 31.25 \text{ kN-m}$ $a_2 = 2 \times 5 \times 31.25/3 = 104.17$ $x_2 = 5/2 = 2.5 \text{ m}$	
		Using three moment theorem;	
		$M_A \times L_1 + 2M_B(L_1 + L_2) + M_C \times L_2 = -[(6 \times a_1 \times x_1/L_1) + (6 \times a_2 \times x_2/L_2)]$	
		M _A = M _C = 0 (End simple supports)	01 M
		$0 + 2M_B(4 + 5) + 0 = -[(6 \times 40 \times 2/4) + (6 \times 104.17 \times 2.5/5)]$	
		$18M_{\rm B} = -(120 + 312.51)$	
		$M_{\rm B} = -432.51 / 18$ = -24.03 kN-m	01 M
Q.4	b)	Using three moments method, find support moments for continuous beam shown in fig. Draw	
Q.4		B. M. D.	
	Ans		





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		6M _A +	$2M_B(6+8) + 0 =$	- [(0) + (6 x 85.33 x 4	l/8)]				
		6M _A +	$28M_B = -(256)$					01 M	
		- 3M _B	+ 28M _B = -256						
		M _B = -	- 256 / 25						
		= -	- 10.24 kN-m						
		M _A = -	- (-10.24/2) = 5.12	2 kN-m.					
				•	16 kN-m				
				10.24 kN-m					
				111111/				01	
							Mark		
		5.1	2 kN-m	B. M. D.					
Q.4	d)	A continuous	beam ABC is simp	oly supported at A, B	and C. Sp	oan AB and span BC a	re of length 5		
				over entire span. Cal	culate su	pport moments by us	sing moment		
		distribution m	nethod.						
	Ans		_30 kN/m						
		m		~~~~					
		A -	5m	B 5m		C			
			2.		2.				
			² /12 = – 62.5 kN-	m $M_{BA} = 30 x$	$5^{2}/12 = 6$	52.5 kN-m		01 M	
		$M_{BC} = M_{BC} = 0$		1	-	1	1		
		Joint	Member	Stiffness (k)	Σ		-		
		В	BA	3 x EI/5 = 0.6EI	1.2	0.6EI/1.2E	1 = 0.5	01 M	
		D	BC	3 x EI/5 = 0.6EI	1.2	0.6EI/1.2E	I = 0.5		
		F							
		Joint		А		B	С		
		Members		AB	BA	BC	CB		
		Dist ⁿ . factor		1.0	0.5	0.5	1.0	02 M	
		F.E.M.		-6 2.5	62.5	0	0		
		Balancing		62.5	-31.25	-31.25	0		
		Carry over			31.25				
		Balancing			-15.625	-15.625			
		Final momer	nts	0.0	46.875	- 46.875	0.0		
		$M_{\rm A} = 0, M_{\rm B} = 4$	16.875 kN-m (Hog	gging) M _c = 0					
Q.4	e)				moment	at fixed end of propp	ed cantilever		
				/m over entire span.					
	Ans			25 kN/m					
			25 kN/m						
1									
		/	A	5 m	В				
		/			В				
		M _{AB} = - 25 x 5	$A^{2}/12 = -52.083 \text{ k}$			= 52.083 kN-m		01 M	
		/ М _{АВ} = – 25 х 5 Јој				= 52.083 kN-m B		01 M	

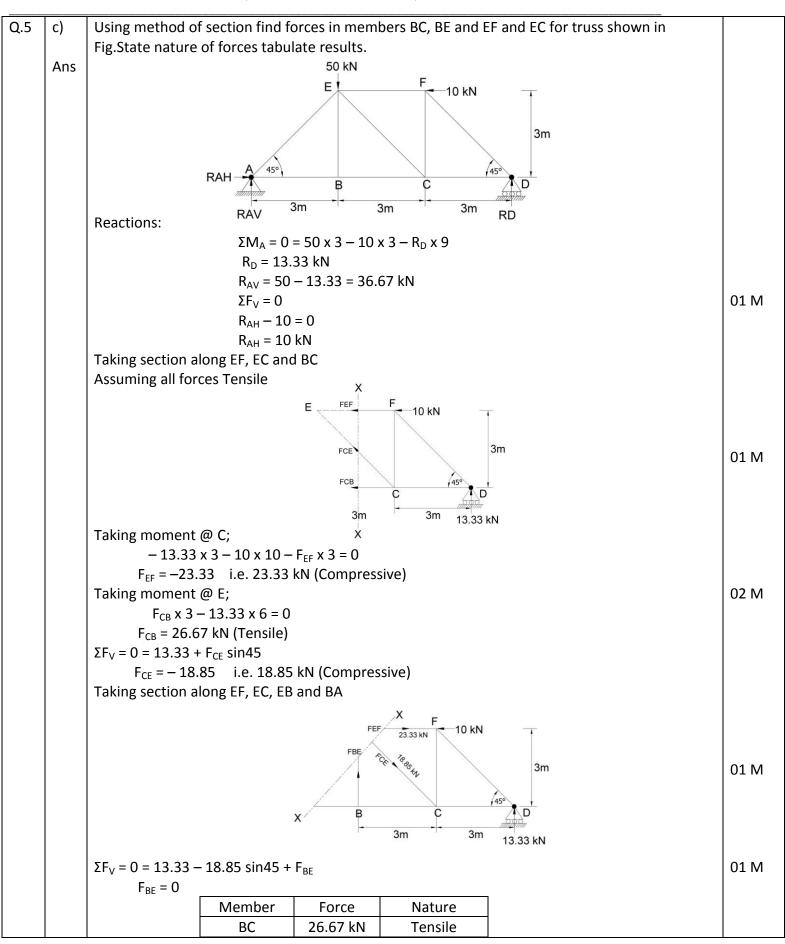


	1		embers	AB			BA		03 M
			st ⁿ . factor	1.0			<u>ВА</u> 1.0		05 101
			E.M.	- 52.	083		52.083		
			llancing		085		- 52.083		
			irry over	- 26.	0/17		- 52.085		
			nal moments	- 78.			0.0		
			kN-m (Hogging)	- 78.	125		0.0		
Q.4	f)		istribution factors a	t continuity fo	r a cont	inuous heam	ABCD which is	fixed at A	
Q.4	''		ed at B, C and D. Tak						
			В		С		D		
		A			4				
		1.5	4m	4m	-1-	5m	-1		
	Ans	Joint	Member	Stiffness	(k)	Σk	D.F. = k	/Σk	
	/ 113	501110	BA	4 x El/4 =	. ,	ZR	EI/2EI =		01 M
		В	BC	$4 \times EI/4 =$		2EI	EI/2EI =		for
			СВ	4 x El/4 =			EI/1.6EI =		each
		C	CD CD	3EI/5 =0.6		1.6EI	0.6EI/1.6EI		factor
			CD	3EI/5 =0.0			0.0EI/1.0EI	= 0.375	lactor
Q.5		Attemnt any	TWO of the followi	nσ					(16)
Q.J	a)	. ,	nimney of uniform h	0					(10)
	Ans	chimney if max. compressive stress at the base is limited to 280 kN/m ² . Also sate nature of minimum stress. Take density of masonry = 22 kN/m ³ . Data: External dimensions = 2.0 m x 1.4 m Internal dimensions = 1.4 m x 0.8 m Horizontal wind pressure (p) = 1.5 kN/m^2 Unit weight of material (σ) = 22 kN/m^3							01 M
		$6_d = \sigma h = 2$	$2 \times h = 22h \text{ kN/m}^2$						
		Case 01:- Lo pressure:	onger face subjected	l to wind		e 02:- Shorter sure:	face subjected	to wind	
		Horizontal v		п x B x h x 2	Hori	zontal wind fo		x B h x 1.4	01 M
		Moment ab	= 3h out base (M) = P x h	•	Mon	nent about ba	= 2.1h se (M) = P x h/		01 M
			= 3h x = 1.5h	•			= 2.1h x = 1.05h	•	
		= 0.3976 m) - (1.4 x 0.8 ³)]/12		= 0	.7504 m ⁴).8 x 1.4 ³)]/12		01 M
		$y_{max} = 0.7$				= 1.0			
		$\int \mathbf{b}_{b} = \mathbf{M} \mathbf{x} \mathbf{y}_{ma}$				$M x y_{max}/I$			01 M
			0.7 / 0.3976			: 1.05h ² x 1.0 / : 1.4h ²	/ 0.7504		
		$= 2.64 h^2$							



			-		-				11
		$6_{max} = 6_d + 6_b$			$6_{max} = 6_d + 6_b$				
		280 = 22h +		$280 = 22h + 1.4h^2$			_		
		h = 6.943 m.			h = 8.32 m.			01 M	
		б _d = 22 x 6.9							
			.943 ² = 127.26 kľ						01 M
		б _{тіп} = 152.74	46 – 127.26 = 25.4	8 kN/m ² $f_{min} = 183.04 - 96.91 = 86.13 \text{ kN/m}^2 \text{ kN/m}^2$			/m² kN/m²		
		(Compressive)			(Compr	ressive)			01 M
Q.5	b)	A continuous	beam ABCD is 15	m long rests on s	upports	A, B and C al	l at same leve	el. AB = 6 m,	
		BC = 5 m, CD = 4 m. It carries two concentrated loads 90 kN and 80 kN at 2 m and 8 m from A							
		respectively a	nd a udl of 30 kN	/m over CD. Find	support	moment by	using momer	nt distribution	
		respectively and a udl of 30 kN/m over CD. Find support moment by using moment distribution method and draw BMD.							
	Ans								
		90	kN	80 KN					
		Α				30 kN/m	ו בייניים ד		
		2m	- 4m	B	C3m	- 4m			
		2111		211	JIII	-111			
		$M_{AB} = -90 \times 2 \times 4^2/6^2 = -80.0 \text{ kN-m}$ $M_{BA} = 90 \times 4 \times 2^2/6^2 = 40.0 \text{ kN-m}$							02 M
		$M_{AB} = -90 \times 2 \times 4 / 6 = -80.0 \text{ kN-m}$ $M_{BC} = -80 \times 2 \times 3^2 / 5^2 = -57.6 \text{ kN-m}$ $M_{BC} = 80 \times 2^2 \times 3 / 5^2 = 38.4 \text{ kN-m}$							02 101
				= -	00 / 2 /	(3/3 - 30.4			
		$\begin{array}{ c c c c c } MCD = -30 \times 4^2/2 = -240.0 \text{ kN-m} \\ \hline Joint & Member & Stiffness (k) & \Sigma k & D.F. = k/\Sigma k \\ \hline \end{array}$						k/5k	
		JOINT				ZK	0.5EI/1.1E		02 M
		B BA		$3 \times EI/6 = 0.5EI$		- 1.1EI			02 101
			BC	3 x EI/5 =0.6			0.6EI/1.1	EI = 0.5	
		Joint A			ВС				
		Joint		AB	BA	BC	СВ	CD	
		Members							
		Dist ⁿ . factor		1.0	0.45	0.55	1.0	0.0	02 M
		F.E.M.		- 80.0	40.0	- 57.6	38.4	- 240.0	02 101
		Balancing		80.0	7.92	9.68	201.6	0.0	
		Carry over Balancing			40.0	100.8			
					-63.36	-77.44			
		Final momer	nts	0.0	24.56	- 24.56	240.0	-240.0	
		240 kN-m							
		120 kN-m						02.14	
		24.56 kN-m 96 kN-m						02 M	
		B. M. D.							







			BE	0			02 M		
			EF	23.33 kN	Compressive		02		
			EC	18.85 kN	Compressive				
Q.6		Attempt any TWO of the following:					(16)		
Q.6	a)	A simply supported beam of span 8 m is subjected to point loads of 60 kN, 80 kN and 50 kN at 2							
	,	m, 4 m and 6 m from left support respectively. Determine slope at left support and deflection							
		under 60 kN and 80 kN loads. EI = $2.668 \times 10^9 \text{ kNm}^2$.							
	Ans								
		60 kN	60 kN 80 kN 50 kN X X 60 kN 80 kN 50 kN						
		RA=97.5 kN							
		$ \underbrace{OR}_{x} \underbrace{2m}_{x} 2$							
		Reactions:							
		$\Sigma M_A = 0$							
		$60 \times 2 + 80 \times 4 + 50 \times 6 - R_B \times 8 = 0$							
		$R_{\rm B} = (120 + 320 + 300) / 8 $							
		= 92.5 kN.							
			+ 80 + 50 – 92.5 = 97.5 kN.						
		Taking section 3			-	n X-X at distance 'X' from B			
		$M_x = 97.5 \times X - (X, C)$	60 x (X-2) – 80	x (X-4) – 50 x		$-50 \times (X-2) - 80 \times (X-4) - 60 \times$	01 M		
		$\left \begin{array}{c} (X-6) \\ EId^2y/dx^2 = -Mx \end{array} \right $			(X-6) Eld ² y/dx ² = - I	My			
		•	^ ′.5 x X + 60 x (X	(-2) + 80 x (X-	• •	92.5 x X + 50 x (X-2) + 80 x (X-			
		4) + 50 x (X-6)	.5 X X * 66 X ()	(2) 00 x (x	4) + 60 x (X-6				
		Integrating			Integrating	,			
		Eldy/dx = -97.5	$x X^{2}/2 + 60 x (2)$	X-2) ² /2 + 80 x	EIdy/dx = -92	$.5 \times X^2/2 + 50 \times (X-2)^2/2 + 80 \times 10^{-10}$	01 M		
		$(X-4)^2/2 + 50 x$	$(X-6)^2/2 + C_1$			$(X-6)^2/2 + C_1$			
		Integrating	(/·····) ³		Integrating	342			
		Ely = $-97.5 \times X^3$				$(x^{3}/6 + 50 \times (X-2)^{3}/6 + 80 \times (X-1)^{3}/6 + $	01 M		
		$(4)^{3}/6 + 50 \times (X-6)^{3}/6 +$	• · · –	C_2	$(4)^{1}/6 + 60 \times (7)^{1}$ At X = 0; y = 0	$(x-6)^3/6 + C_1 \times X + C_2$			
		A = 0, y = 0 $0 = 0 + C_2$	in ciy eq .		A = 0, y = 0 $0 = 0 + C_2$, in Eiveq .			
		$C_2 = 0$			$C_2 = 0$				
		At X = 8; y = 0	in Ely eq ⁿ .		At X = 8; y = 0) in Ely eq ⁿ .			
		$0 = -97.5 \times 8^{3}/6$	$+60 \times (8-2)^{3}/6$	5 + 80 x (8-	$0 = -92.5 \times 8^{3}$	$(6 + 50 \times (8-2)^3/6 + 80 \times (8-2)^3/6 \times (8-2)^3/6 + 80 \times$			
		4) ³ /6 + 50 x (8-6	5) ³ /6 + C ₁ x 8 +	0	4) ³ /6 + 60 x (8	$(3-6)^3/6 + C_1 \times 8 + 0$			
		C ₁ = 655			C ₁ = 645		01 M		
		Hence $C_1 = 655$	-		Hence $C_1 = 64$	_			
		Slope equation		$0 \times (1 \times 2)^2$	Slope equation				
		dy/dx =(1/EI)[- 80 x (X-4) ² /2 + 5		• • •		$[-92.5 \times X^2/2 + 50 \times (X-2)^2/2 + 60 \times (X-6)^2/2 + 645](01)$			
		Deflection equal		000](01)	Deflection eq		01 M		
		y =(1/EI)[-97.5		X-2) ³ /6 + 80 ×		$2.5 \times X^3/6 + 50 \times (X-2)^3/6 + 80 \times 10^{-10}$	<u><u><u></u></u></u>		
		$(X-4)^3/6 + 50 \times ($				$x (X-6)^{3}/6 + 645 x X$](02)			
		For slope at sup			For slope at s				
		Put X = 0 in eq ⁿ	.01		Put X = 8 in e	q ⁿ .01			



				-				
		$(dy/dx)_A = (1/EI) x (655) = 655 / EI$	$(dy/dx)_{A} = (1/EI)[-92.5 \times 8^{2}/2 + 50 \times (8-2)^{2}/2$					
		$= 655 / 2.668 \times 10^9 = 2.455 \times 10^{-7}$ rad.	$+80 \times (8-4)^2/2 + 60 \times (8-6)^2/2 + 645$]					
		For deflection at B	$= 655 / 2.668 \times 10^9 = 2.455 \times 10^{-7}$ rad.	01 M				
		Put $X = 2$ in eq ⁿ .02	For deflection at B	•= ···				
		$y_{\rm B} = (1/EI)[-97.5 \times 2^3/6 + 655 \times 2]$						
			Put X = 6 in eq ⁿ .02					
		$= 1180 / 2.668 \times 10^9 = 4.423 \times 10^{-7} \text{ m.}$	$y_{\rm B} = (1/EI)[-92.5 \times 6^3/6 + 50 \times (6-2)^3/6 + 80$					
		= 4.423 x 10 ⁻⁴ mm.	$x (6-4)^3/6 + 645 \times 6$					
		For deflection at C	= 1180 / 2.668 x 10 ⁹ = 4.423 x 10⁻⁷ m.					
		Put X = 4 in eq^{n} .02	= 4.423 x 10 ⁻⁴ mm.					
		$Y_{\rm C} = (1/EI)[-97.5 \times 4^3/6 + 60 \times (4-2)^3/6 + 655$	For deflection at C	01 M				
		[x4]	Put X = 4 in eq^{n} .02					
		$= 1660 / 2.668 \times 10^9 = 6.222 \times 10^{-7} \text{ m.}$	$Y_{c} = (1/EI)[-92.5 \times 4^{3}/6 + 50 \times (4-2)^{3}/6 + 645]$					
		$= 100072.008 \times 10^{-9} = 0.222 \times 10^{-4}$ mm.						
		= 6.222 x 10 mm.	x 4]					
			= 1660 / 2.668 x 10^9 = 6.222 x 10^{-7} m.					
			$= 6.222 \text{ x } 10^{-4} \text{ mm.}$					
Q.6	b)	A fixed beam AB of span 6 m carries point loac	ls of 120 kN and 90 kN at 2 m and 4 m from left					
	-	hand support. Find fixed end moments and su						
	Ans	120 kN S	90 kN					
		A	B					
		MAB RA 2m 2m	2m MBA					
		KB						
		$M_{AB} = (120 \times 2 \times 4^2 / 6^2) + (90 \times 4 \times 2^2 / 6^2)$						
		= 146.67 kN-m						
		$M_{BA} = (120 \times 2^2 \times 4 / 6^2) + (90 \times 4^2 \times 2 / 6^2)$						
		$M_{BA} = (120 \times 2 \times 4 / 6) + (90 \times 4 \times 2 / 6)$ = 133.33 kN-m						
		Reactions:						
		$\Sigma M_A = 0$						
		$120 \times 2 + 90 \times 4 + 133.33 - 146.67 - R_B \times 6 = 0$						
		$R_B = (240 + 360 - 13.33) / 6$						
		= 97.78 kN.						
		$R_A = 120 + 90 - 97.78 = 112.22 \text{ kN}.$						
		Bending moment at point load						
		$M_{\rm C} = -146.67 + 112.22 \times 2$						
		= 77.77 kN-m						
		$M_{\rm D} = -146.67 + 112.22 \text{ x} 4 - 120 \text{ x} 2$						
		= 62.21 kN-m						
		Shear force calculations:						
		At B = – 97.78 kN						
		At D, just right = –97.78 kN						
		At D, just left = $-97.78 + 90 = -7.78 \text{ kN}$						
		$\Delta t C$ just right = -7.78 kN						
		220 Kithi 200 kN m	-					
		$\begin{array}{c} 200 \text{ NV-m} \\ 146.67 \text{ kN-m} \end{array} \qquad \begin{array}{c} \text{At C, just left} = -7.78 + 120 = 112.22 \text{ kN} \\ 133.33 \text{ kN-m} \end{array}$						
		At A = 112.22 kN						
		B. M. D.						
				1				



