

WINTER-18 EXAMINATION

Subject Name: Thermal Engineering

Model Answer

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17410
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Important Instructions to examiners:

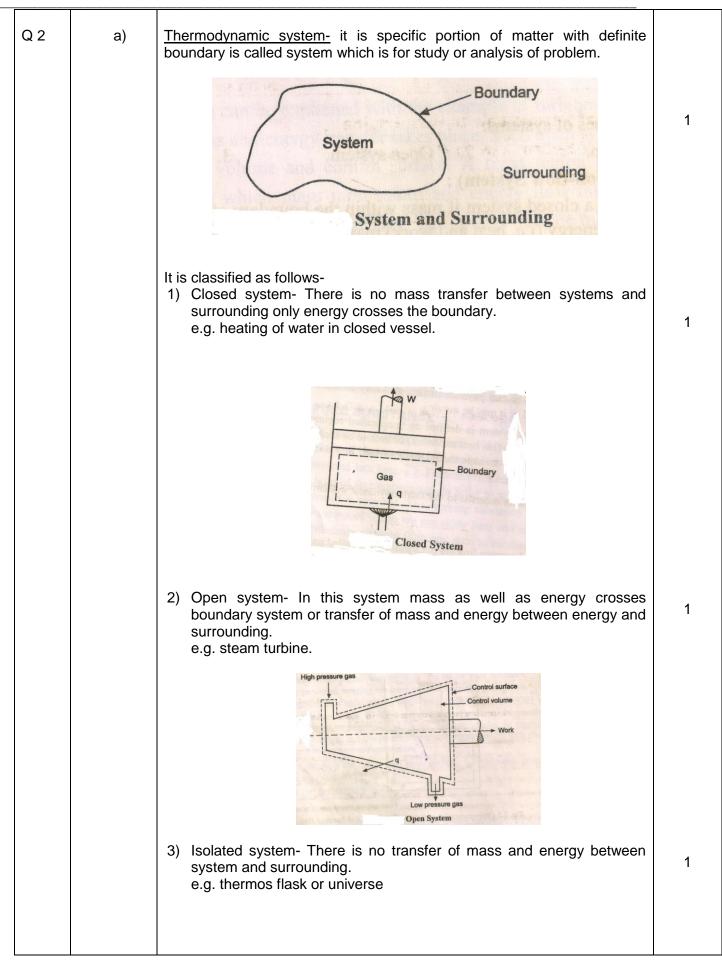
- 1) The answers should be examined by key words and not as word-to-word as given in the model answer scheme.
- 2) The model answer and the answer written by candidate may vary but the examiner may try to assess the understanding level of the candidate.
- 3) The language errors such as grammatical, spelling errors should not be given more Importance (Not applicable for subject English and Communication Skills.
- 4) While assessing figures, examiner may give credit for principal components indicated in the figure. The figures drawn by candidate and model answer may vary. The examiner may give credit for any equivalent figure drawn.
- 5) Credits may be given step wise for numerical problems. In some cases, the assumed constant values may vary and there may be some difference in the candidate's answers and model answer.
- 6) In case of some questions credit may be given by judgement on part of examiner of relevant answer based on candidate's understanding.
- 7) For programming language papers, credit may be given to any other program based on equivalent concept.

Q.NO.	Sub. Q.No.	Answer	Marking Scheme
Q.1(a)	i)	<u>Intensive Property</u> - The properties are independent on mass of system is known as Intensive Properties. Example-Pressure, Temperature.	2
	ii)	<u>Kelvin Plank Statement-</u> It is impossible to construct a heat engine which while operating in a cyclic process, will produce no effect other than transfer heat from single thermal reservoir and performance an equivalent amount of work.	2
	iii)	<u>Boyle's Law-</u> When a perfect gas is heated at constant temperature, the volume of given mass of gas is inversely proportional to absolute pressure. V α 1/P, PV=C.	2
	iv)	Isothermal Process- ₽↑	2
		$P_{1} \xrightarrow{P_{1}} Constant temperature process (p.v = constant)}{P_{2} \xrightarrow{V_{1}}} \xrightarrow{V_{2} \xrightarrow{V_{2}}} V$	
	v)	<u>Boiler Mounting</u> -These are fitting or mounting on boiler for safety of boiler and for complete control over the processes of steam generation. a) Safety valve b) water level indicator c) Fusible plug d) pressure gauge e) steam stop valve f) feed check valve (Any Two)	2

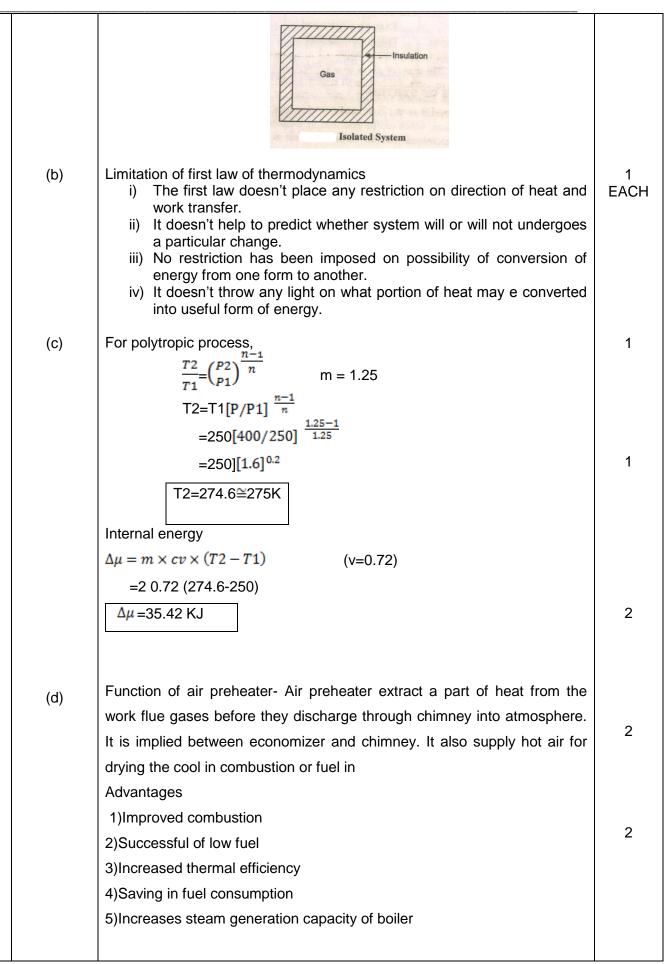


vi) Four losses in steam turbine(any four) 1) Friction losses 2) Leakage losses 2 3) Exhaust losses 4) Radiation and convection losses 5) Losses due to moisture 6) Admission loss 7) Carry over loss Condenser efficiency- It is defined as 'the ratio of rise in temperature of vii) cooling water to the difference between the saturation temperature to 2 absolute pressure in condenser and inlet temperature of cooling water. ' Or It is defined as the ratio of actual rise in the temperature of cooling water to the maximum possible rise. Fourier law of heat conduction- It state that 'for homogeneous material, 2 viii) the rate heat transfer in steady state in any direction is linearly proportional to temperature gradient in that direction. ' (b) i) 1) Dryness fraction- is define as,'the amount of dry steam in one kg of wet EACH steam.' Denoted by 'x' 1 2) Enthalpy of wet steam- It is quantity of heat required to convert 1 kg of water at 0c. in wet steam having dryness fraction x at constant pressure. 3) Enthalpy of dry saturated steam- the quantity of heat required to convert 1 kg of water at 0c into dry saturated steam at constant pressure. 4) Enthalpy of superheated steam- The quantity of heat required to convert 1 kg of air at 0c into superheated steam. ii) Function of cooling tower is that hot water coming out from condenser is 2 feed to the tower on top and allow to tickle in form of thin sheets or drop. Air flows from bottom of tower or perpendicular to direction of water flow and then exhausts to atmosphere after effective cooling of water. Uses-1) For cooling hot water from condenser in thermal power plant. 2) For supply of water to steam power plant when less water is available. 2 Heat exchanger are classified as 1) Direct contact or open heat exchanger iii) 2) In direct contact 2 3) Parallel flow 4) Counter flow 5) Concentric flow 6) Shell and tube type 7) Multiple shell and tube passes Application-1) In refrigerating system 2) Radiator of automobile 3) Solar water system 2 4) Regenerators 5) Milk chiller of pasteurising plant











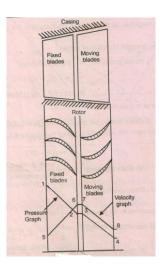
(e)

MAHARASHTRA STATE BOARD OF TECHNICAL EDUCATION (Autonomous) (ISO/IEC - 27001 - 2013 Certified)

Working principle of reaction turbine-

-A turbine in which steam pressure decreases gradually while expanding through the moving as well as through the fixed blade is known as reaction turbine.

-In pure reaction turbine , the drop of pressure with expansion and generation of kinetic energy take place in moving blades the steam jet leaves the moving blades at greater velocity than that they enter blades. The jet of steam leaving the moving blade with greater velocity reacts on the blades and turn them round. The passage through moving blade of reaction turbine is made convergent so the steam expand as is passes through moving blades. The expansion causes the steam to leave the moving blade as higher velocity than as which it entered.



Working of regenerative feed heating system-

(f)

Turbine Boiler Exhaust steam with Bleed berhe Condense stean www. Outlet Feed water heater Feed Cooling wate Bleed steam condensate -The process of draining steam from turbine at certain points during it's expansion and using this steam for heating feed water supplied to boilerisknown as regenerative feed heating.

2

2

1

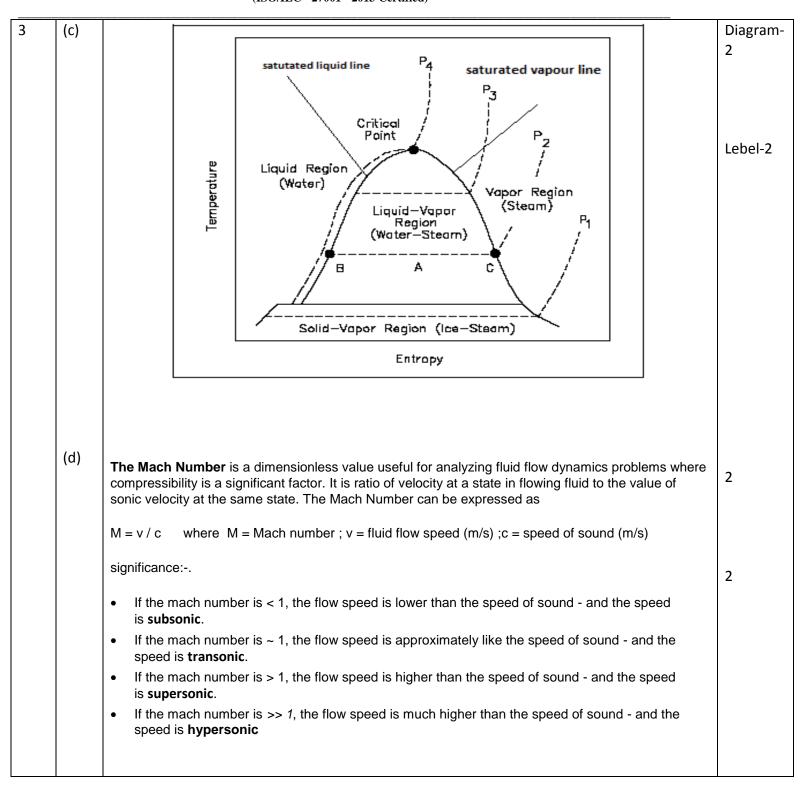
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<u>T</u>	wo advantages- i) It increases the thermal efficiency of plant.	
	 ii) The temperature stresses in the boiler are reduced due to decreased range of working temperature. 	1

Q.	Sub	Answer	Marking
No.	Q. N.		Scheme
Q3	(a)	Attempt any FOUR	
		A heat pump is a thermodynamic system operating in a cycle, which removes heat from low temperature body and delivers it to high temperature body.	1
		Heat pumps move thermal energy in the opposite direction of spontaneous heat transfer, by absorbing heat from a cold space and releasing it to a warmer one.	1
		Refrigerator: Heat is withdrawn from inside a refrigerator (low temperature). Temperature inside refrigerator drops. Extracted heat is given off to environment (higher temperature). Negligible temperature rise of the environment. (COP) _R = Q2/(Q1-Q2) Heat pump: Heat is withdrawn from environment: air, soil, or water (low temperature). Negligible temperature drop of the environment. Extracted heat is given off into building (higher temperature). Temperature of building's interior rises. (COP) _p = Q1/(Q1-Q2) ; (COP) _p = 1 + (COP) _R	1
	(b)	Isobaric process (Constant pressure process) Work done W1-2 = P(V2-V1) P= pressure ; V1 and V2 are initial and final volumes	2
		Heat transfer Q1-2 = m Cp(T_2 - T_1) m= mass; Cp is specific heat at constant pressure; T_1 and T_2 ate initial and final temp.	2





Q. No.	Sub Q. N.	Answer	Marking Scheme
NO.	Q. N.		Scheme
	(e)	The sources of air in the condenser are due to the following reasons:- i. Leakage through packing glands and joints.	2
		ii. Leakage through condenser accessories, such as atmospheric relief value, etc.iii. Air associated with exhaust steam may also liberate at low pressure.	2
		iv. In the jet condenser, the dissolved air in the cooling water liberates at low	



(f)	 pressures. The effect of the presence of air in condenser are: The pressure in the condenser is increased; this reduces the work done by the engine or turbine. The partial pressure of steam and temperature are reduced. The steam tables tell us that at lower pressure, the latent heat of steam is more. To remove this greater quantity of heat, more cooling water has been supplied and, thus under cooling of the condensate is likely to be more severe resulting in lower overall efficiency. The presence of air reduces the rate of condensation of steam since the abstraction of heat by the circulating cooling water is partly from the steam and partly through the air. The rate of heat transfer from the vapor is reduced due to the poor thermal conductivity of air. Thus, the surface is of the tubes has to be increased for a given condenser duty. An air extraction pump is needed to remove air still some quantity of steam escapes with the air even after shielding to the air extraction section. This reduces the amount of condensate. Moreover, the condensate is under cooled, with the result that more heat has to be supplied to the feed water in the boiler. Shell and Coil type heat exchanger :- (Note –Sketch of shell and tube type may also be accepted, since it also contains coils, and also mechanism of heat transfer is similar) Here, thermal energy between two fluids at different temperatures is transferred. Both fluids are in motion and main mode of heat transfer is convection. Here, bundle of round tubes are placed in cylindrical shell . The tube bundle may be parallel to axis (shell and tube), or helical coil (shell and coil). Generally cross flow pattern is used, which gives better heat transfer.	2
	SHELL INLET CORE TUBE TUBE OUTLET	2



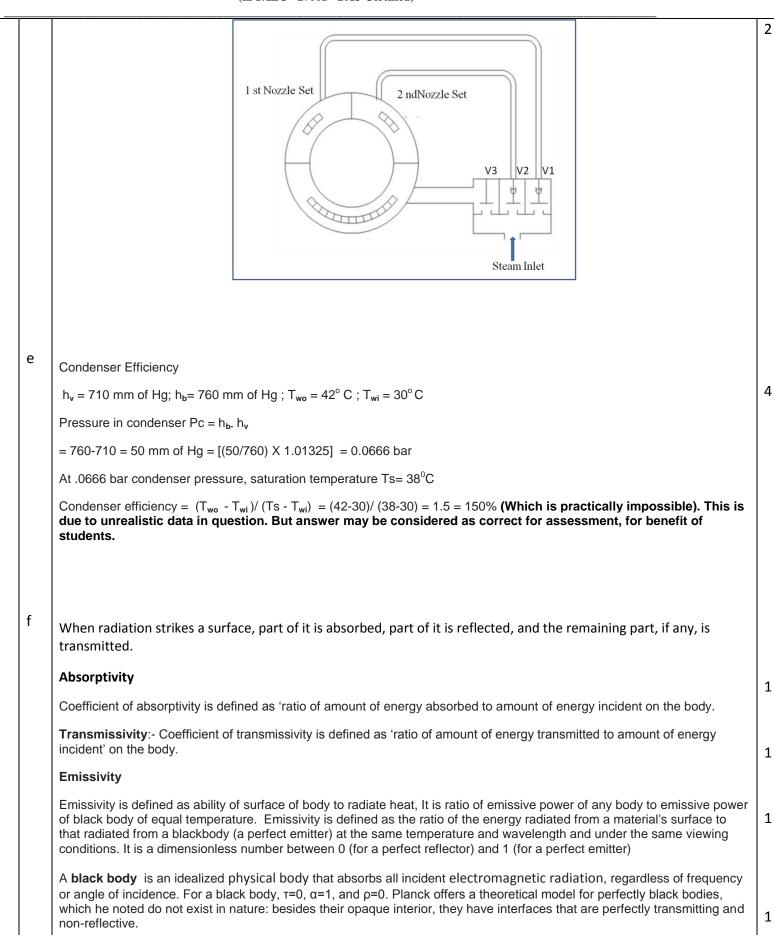
Q4	(a)	(i)State-It may be identified or described by some observable quantities such as volume, pressure, temp, density, which are thermodynamic properties. Minimum two properties are required to define state of system. Properties are state or point functions.	1
		(ii) Process:-When system changes its state from one equilibrium state to another equilibrium state, then the path of successive states through which, the system has passed is known as thermodynamic process.	1
		(iii)Point function: - If each property has single value at each state i. e. properties of system depends upon state of system. These properties are called as point function. e.g. pressure, temp.	1
		(iv) Path function: - The thermodynamic quantities, which are dependent on path followed between two end states of process are path functions. e.g. Work, heat	1

4	(b)	Mass of steam = 4 kg	1
4		Pressure = 2MPa = 20 bar	1
		From steam tables, Enthalpy hf= 908.69 KJ/kg; hfg= 1890.01 KJ/Kg; hg = 2798.8 KJ/kg;	
		s1= 2.447 KJ/kg K; sg = 6.3396 KJ/kg K ;Vg = 0.09959 m3/Kg	
		Enthalpy of dry saturated steam hg = 2798.8 KJ/kg	1
		Enthalpy of 4 kg of steam = 4 X 2798.8 = 11195.2 KJ/Kg	
		Specific volume = Vg = 0.09959 m3/Kg	1
		Volume of 4 kg of steam = 4 X 0.09959 = 0.398 m3	
		Specific Entropy = 6.3396 KJ/kg K	
		Entropy of 4 kg of steam = 25.32 KJ/K	
		Internal energy of 1 kg of dry saturated steam	
		u=hg-Pvg	1
		=2798.8-[(20 X 10 ⁵ /1000) X 0.09959)]	
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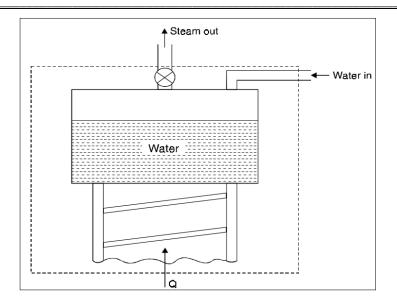
	= 2599.62KJ/kg	
	Internal energy of 4 kg of dry saturated steam =10398.48 KJ	
с	Principle of Forced Draught	
	Fan or blower is placed before grate or situated within the oil burner assembly in case of oil fired boiler.	1
	Pressure inside the flue passages is slightly more than atmospheric pressure.	
	Due to above reason, there is serious danger of fire to come out and injure someone if any leakage in boiler flue passage take place.	
	It forces fresh air into the combustion chamber, which helps in burning of fuel and production of hot combustion gases. Due to this force, the hot flue gases are further pushed through the flue passages in boiler.	
	Advantages:-Forced draught fan require less power because it has to handle cold and dense air, the volume per unit mass of which is less.	
	Flow of flue gases through boiler is more uniform.	1
	Principle of Induced Draught	
	The fan or blower is placed after the grate or after the flue passages and before chimney.	1
	Pressure inside the flue passages is slightly less than atmospheric pressure.	
	It sucks the hot flue gases from the combustion chamber through flue passages and then passes on these to economizer, air pre heater and chimney. Due to this suction, fresh air is also sucked in to combustion chamber.	
	Advantage: - Due to above reason, there is no danger of fire to come out from boiler flue passage in case of a leakage. Hence Induced draught is safer.	1
d	Nozzle control Governing in steam turbine-	
	In nozzle governing the flow rate of steam is regulated by opening and shutting of sets of nozzles rather than regulating its pressure. In this method groups of two, three or more nozzles form a set and each set is controlled by a separate valve. The actuation of individual valve closes the corresponding set of nozzle thereby controlling the flow rate. In actual turbine, nozzle governing is applied only to the first stage whereas the subsequent stages remain unaffected. Since no regulation to the pressure is applied, the advantage of this method lies in the exploitation of full boiler pressure and temperature. Figure shows the mechanism of nozzle governing applied to steam turbines.	2













2) Nozzle is a device which increases velocity
of worthy substance.
• The nozzle is insulated so that no hedd
transfer takes place in system.
i.e.
$$\underline{P_{1-2} = 0}$$

• The nozzle does not deliver any work
i.e. $\underline{W_{1-2} = 0}$
• There is no change in potential energy
i.e. $\underline{P_{e2} - P_{e_1} = 0}$
we know sfee for unit mass flow is
 $q_{1-2} - W_{1-2} = (h_2 - h_1) + (ke_2 - ke_1) + (fe_2 - fe_1)$
 $0 = (h_2 - h_1) + (ke_2 - ke_1) + (fe_2 - fe_1)$
 $0 = (h_2 - h_1) + (ke_2 - ke_1) + (fe_2 - fe_1)$
 $v_2^2 - v_1^2 = 2(h_1 - h_2)$
 $v_2^2 - v_1^2 = 2(h_1 - h_2)$
 $v_2^2 = 2(h_1 - h_2) + v_1^2$
Steam of
 $v_1 = \sqrt{2(h_1 - h_2) + v_1^2}$

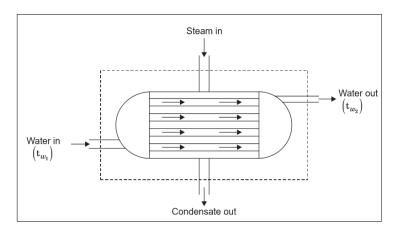


3) Turbine A turbine is a device which converts energy of a working substance into work in turbine . The turbine is insulated so that there is no heat transfer take place. i.e. 91-2=0 · The kinetic energy of Potential Energy are negligible. i.e. [Kez-Ke1=0], [Pez-Pe1=0] we know SFEE for unit mass flow is 91-2 - W1-2 = (h2-h1) + (ke2-kei) + (le2-le1) $-W_{1-2} = h_2 - h_1$ $|W_{1-2} = h_1 - h_2|$ This shows that working work is done by the system due to decrease in enthalpy. Gas or steam in Boundary (1)Generator Turbine Gas or steam out

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4) condenser A condenser is a device used to condense Steam in care of power plants using water as a cooling medium. · There is no change in Kinetic energy f potential energy i.e. [Kez-Ke1=0], [Pez-Pe1=0] There is no work done by the oysten. i.e. | W1-2 =0 we know SFEF for whit mass flow is 91-2 - W1-2 = (h2-h1) + (Kez-kei) + (Pez-Pei) $-9_{1-2} = ha - h_1$ as the heat is rejected from condenser to surrounding, the 91-2 is negative 9,-2 = h,-h2





$V_{i} = s \cdot \sum m^{3}$	
$P_1 = 5 bar.$	
$T_1 = 180^\circ c = 180 + 273 = 453 k$	4
$P_1 = P_2$	
$ \rightarrow $	
V S	2
> To find change in inte	nal Energy
$\Delta u = m cv (T_2 - T_1)$	
find m & T2	
$\frac{V_1}{\cdots} = \frac{T_1}{\cdots}$	
$T_2 = T_1 \times \frac{\sqrt{2}}{\sqrt{2}}$	
2 1 5 2 1 2 2	
$-433 \times \frac{29}{V_1}$	
T2=906K	
	그는 것 같아. 것
	$T_{1} = 180^{\circ}c = 180 + 273 = 4531$ $F_{1} = F_{23}$ $V_{2} = 2V_{1}$ $V_{2} = 2V_{1}$ $V_{2} = 0$ $P = 1 T = 1 T^{2}$ $T = 1 T^{2}$ $F = 1 T^{2}$



To find m, take ideal gaz eqⁿ

$$\begin{bmatrix} PV = mRT \end{bmatrix}$$
use it for initial state
 $P_{1}V_{1} = mRT_{1}$
find out R
 $ar [CP - CW = R]$
 $1 - 0.715 = R$
 $R = 0.285 \text{ kJ/kgk}$
 $P_{1}V_{1} = mRT_{1}$
 $ar Rig in KJ/kgK$, divide pressure
by 1K or 10³
 $68 \times 10^{5} \times 3.5 = m \times 0.285 \times 453$
 $\boxed{m = 16.25 \text{ kg}}$
 $M = mCv (T_{2} - T_{1})$
 $= 16.25 \times 0.715 (906 - 4.53)$
 $\boxed{\Delta U = 5263.30 \text{ KJ}}$

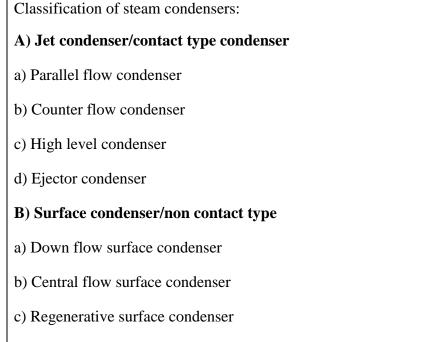
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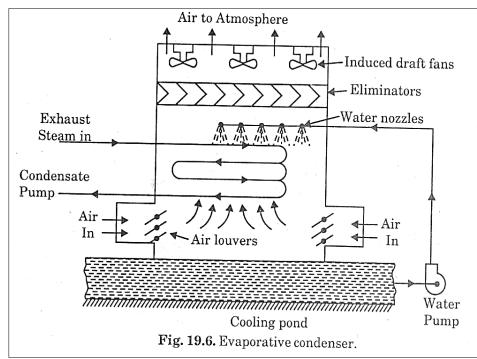
2) work Transferred $W = P(V_2 - V_i)$ = mR (T, -T) = 16.25 × 0.285 (906-453) W= 2097.95 KJ 3) Heat transferred Q= Du +W = 5263.30 + 2097.95 9 = 7361.25 KJ change in Entropy 4) AS= Cpla (T2/T,) = 1 x ln (906/453) DS= 0.6931 KJ/kgk



С



- d) Evaporative surface condenser
- e) Double pass surface condenser or shell and tube type



Working of evaporative condenser.

The refrigerant flows through the coil of the evaporative condenser. Heat from the refrigerant is rejected through the coil tubes. Part of the heat is removed directly by the downward induced air and discharged to the surrounding. Rest of the heat is rejected to the water cascading down over the tubes. Simultaneously, air is drawn in through the air inlet louvers at the base of the evaporative condenser. A small portion of the water is evaporated which removes the heat. The warm saturated air travels through the drift eliminator &

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6 a

discharged by the fan to the surrounding, thereby reducing drift water loss. Post heat exchange, the condensed refrigerant flows to receiver tank.

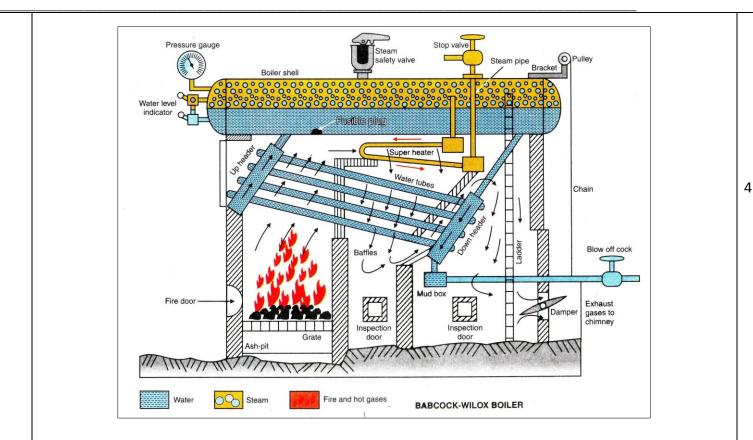
Babcock and Wilcox boiler

Coal is fed to the grate through the fire door and is burnt.

Flow of flue gases: The hot flue gases rise upward and pass across the left-side portion of the water tubes. The baffles deflect the flue gases and hence the flue gases travel in the zig-zag manner (i.e., the hot gases are deflected by the baffles to move in the upward direction, then downward and again in the upward direction) over the water tubes and along the superheater. The flue gases finally escape to atmosphere through chimney.

Water circulation: That portion of water tubes which is just above the furnace is heated comparatively at a higher temperature than the rest of it. Water, its density being decreased, rises into the drum through the uptake-header. Here the steam and water are separated in the drum. Steam being lighter is collected in the upper part of the drum. A continuous circulation of water from the drum to the water tubes and water tubes to the drum is thus maintained. The circulation of water is maintained by convective currents and is known as "natural circulation". A damper is fitted to regulate the flue gas outlet and hence the draught. The boiler is fitted with necessary mountings. Pressure gauge and water level indicator are mounted on the boiler at its left end. Steam safety valve and stop valve are mounted on the top of the drum. Blow-off cock is provided for the periodical removed of mud and sediments collected in the mud box.





Compounding of steam turbines:- If entire pressure drop from boiler pressure to condenser pressure is carried out in a single stage of nozzle then the velocity of steam entering the turbine blades will be very high. The turbine speed has to be also very high as it is directly proportional to steam velocity. Such high rpm of turbine rotor are not useful for practical purposes & there is a danger of structural failure of blades due to excessive centrifugal stresses. Hence compounding is carried out.

Compounding of steam turbines is done:

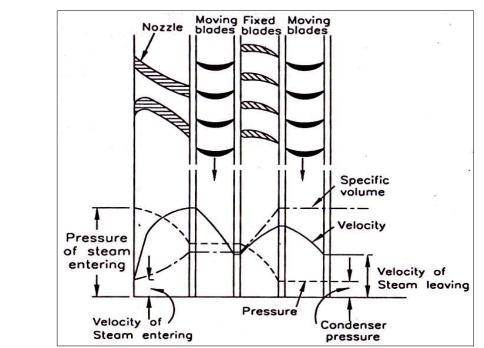
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- \checkmark To reduce speed of rotor blades to practical limits.
- \checkmark To reduce centrifugal force and hence to prevent failure of blades.
- \checkmark To reduce velocity of steam leaving blades.

Pressure compounding of steam turbines. A number of simple impulse turbine stages are arranged in series as shown in fig.

The turbine is provided with one row of fixed blade (nozzle) at the entry of each row of moving blades. The total pressure down of stream does not take place in a single nozzle but is divided among all the rows of fixed blades which work as nozzles.





Pressure compounding of steam turbine



